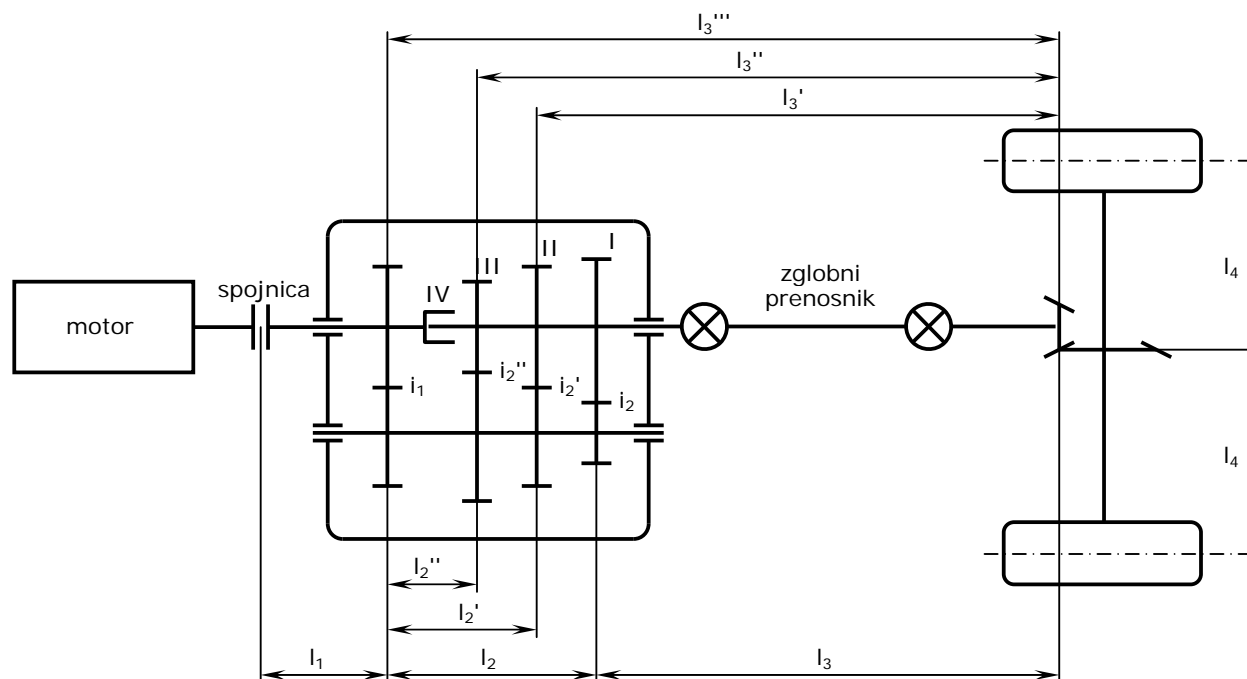
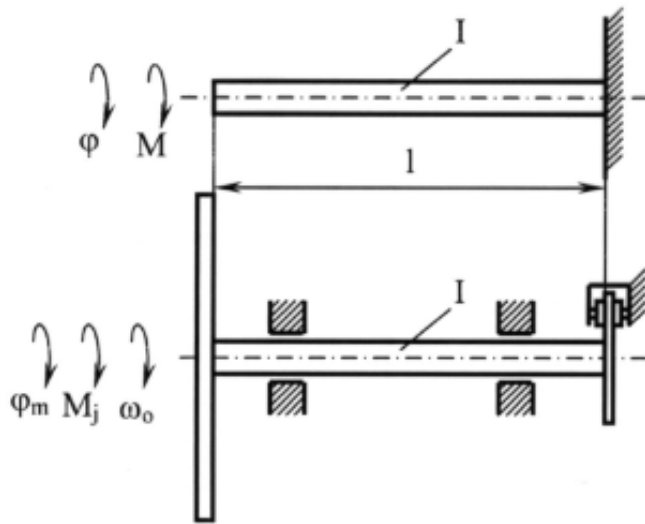


# ODREĐIVANJE UTICAJA INERCIONIH MOMENATA KOD DELOVA I SKLOPOVA MOTORNIH VOZILA

Opterećenja delova i sklopova motornih vozila mogu u određenim slučajevima znatno da premaše opterećenja koja potiču neposredno od pogona.

<p style="background-color: yellow; display: inline-block; margin: 0;"><b>Momenta nošenja spojnice</b></p> $M_n = M_{e_{max}} + J_e \frac{d\omega_e}{dt} = M_{e_{max}} \cdot \beta$	<p style="background-color: cyan; display: inline-block; margin: 0;"><b>Blokiranje pogonskih točkova sa uključenom glavnom spojnicom</b></p> $M_j = J_m \frac{d\omega_m}{dt}$
<p style="text-align: center;">gde je:</p> <p style="text-align: center;"> <math>J_m</math> - moment inercije zamajca motora;  <math>\omega_m</math> - ugaona brzina zamajca motora;  <math>\frac{d\omega_m}{dt}</math> - ugaono usporenje zamajca motora.         </p>	





$$\varphi = \frac{M \cdot l}{G \cdot I} ; \quad \varphi - \text{ugao uvijanja}$$

$$\varphi_m = \frac{M_j \cdot l}{G \cdot I} ; \quad \varphi_m - \text{ugao uvijanja zamajca motora}$$

$$I = \frac{\pi \cdot d^4}{32} ; \quad \text{Moment inercije}$$

$$G = 8.1 \cdot 10^5 \text{ daN/cm}^2 \quad \text{Modul klizanja}$$

$$\varphi = \frac{M \cdot l}{G \cdot I} \quad \varphi_m = \frac{M_j \cdot l}{G \cdot I} \quad \varphi_m = \frac{M_j}{c} \quad c = \frac{GI}{l}$$

$$M_j = J_m \frac{d\omega_m}{dt} \quad M_j = J_m \frac{d\omega_m}{dt} \cdot \frac{d\varphi_m}{d\varphi_m} = c \cdot \varphi_m \quad \Rightarrow \quad c \cdot \varphi_m = J_m \frac{d\varphi_m}{dt} \cdot \frac{d\omega_m}{d\varphi_m}$$

$$\frac{d\varphi_m}{dt} = \omega_m \quad \Rightarrow \quad c \cdot \varphi_m = J_m \cdot \omega_m \cdot \frac{d\omega_m}{d\varphi_m} \quad c \cdot \varphi_m \cdot d\varphi_m = J_m \cdot \omega_m \cdot d\omega_m$$

Integraljenem za početne uslove:

$$\varphi(0) = 0 \quad ; \quad \varphi(t) = \varphi_{\max}$$

$$\omega(0) = \omega_0 \quad ; \quad \omega(t) = 0$$

$$c \cdot \int_0^{\varphi_{\max}} \varphi_m \cdot d\varphi_m = J_m \cdot \int_{\omega_0}^0 \omega_m \cdot d\omega_m$$

$$\frac{1}{2} \cdot c \cdot \varphi_m^2 \Big|_0^{\varphi_{\max}} = \frac{1}{2} \cdot J_m \cdot \omega_m^2 \Big|_{\omega_0}^0$$

$$c \cdot \varphi_{\max}^2 = -J_m \cdot \omega_0^2$$

$$\varphi_{\max} = \omega_0 \cdot \sqrt{\frac{J_m}{c}}$$

$$\rightarrow M_{\max} = c \cdot \varphi_{\max} \quad ; \quad \frac{M_{\max}}{M_e} \quad \rightarrow \text{Analiza čvrstoće}$$

$$M_{\max} = \omega_{\max} \sqrt{J_m c}$$

$$\Rightarrow M_{\max j} = \omega_0 \cdot \sqrt{c \cdot J_m}$$

Pri blokiranju pogonskih točkova, **zamajac motora se zakreće za ugao  $\varphi_m$**  (ako spojnica nije isključena), uvijajući sve delove sistema za prenos snage.

$$\varphi_m = \varphi_1 + \varphi_2 \cdot i_1 + \varphi_3 \cdot i_1 \cdot i_2 + \varphi_4 \cdot i_1 \cdot i_2 \cdot i_o$$

gde je:

$\varphi_1$  - ugao uvijanja izlaznog vratila menjača

$\varphi_2$  - ugao uvijanja posrednog vratila menjača

$\varphi_3$  - ugao uvijanja izlaznog vratila sa zglobnim prenosnikom

$\varphi_4$  - ugao uvijanja pogonskih poluvratila

$$\varphi_1 = \frac{M_j \cdot l_1}{G \cdot I_1} \quad \varphi_2 = \frac{M_j \cdot l_2}{G \cdot I_2} \cdot i_1 \quad \varphi_3 = \frac{M_j \cdot l_3}{G \cdot I_3} \cdot i_1 \cdot i_2 \quad \varphi_4 = \frac{M_j \cdot l_4}{2 \cdot G \cdot I_4} \cdot i_1 \cdot i_2 \cdot i_o$$

$$\varphi_m = \frac{M_j \cdot l_1}{G \cdot I_1} + \frac{M_j \cdot l_2}{G \cdot I_2} \cdot i_1^2 + \frac{M_j \cdot l_3}{G \cdot I_3} \cdot i_1^2 \cdot i_2^2 + \frac{M_j \cdot l_4}{2 \cdot G \cdot I_4} \cdot i_1^2 \cdot i_2^2 \cdot i_o^2$$

Možemo napisati i na sledeći način:

**c - konstanta krutosti transmisije**

$$\varphi_m = \frac{M_j}{G} \left( \frac{l_1}{I_1} + \frac{l_2}{I_2} \cdot i_1^2 + \frac{l_3}{I_3} \cdot i_1^2 \cdot i_2^2 + \frac{l_4}{I_4} \cdot i_1^2 \cdot i_2^2 \cdot i_o^2 \right) \quad c = \frac{G}{\left( \frac{l_1}{I_1} + \frac{l_2}{I_2} \cdot i_1^2 + \frac{l_3}{I_3} \cdot i_1^2 \cdot i_2^2 + \frac{l_4}{I_4} \cdot i_1^2 \cdot i_2^2 \cdot i_o^2 \right)}$$

$$\varphi_m = \frac{M_j}{c} \Rightarrow M_j = \varphi_m c$$

$$M_j = J_m \frac{d\omega_m}{dt} = \varphi_m c$$

$$M_j = J_m \frac{d\omega_m}{dt} \cdot \frac{d\varphi_m}{d\varphi_m} = c \cdot \varphi_m \quad \Rightarrow \quad c \cdot \varphi_m = J_m \frac{d\varphi_m}{dt} \cdot \frac{d\omega_m}{d\varphi_m}$$

$$\frac{d\varphi_m}{dt} = \omega_m \quad \Rightarrow \quad c \cdot \varphi_m = J_m \cdot \omega_m \cdot \frac{d\omega_m}{d\varphi_m}$$

$$\underline{c \cdot \varphi_m \cdot d\varphi_m = J_m \cdot \omega_m \cdot d\omega_m}$$

Integraljenjem za početne uslove:

$$\begin{aligned} \varphi(0) = 0 & \quad ; \quad \varphi(t) = \varphi_{\max} \\ \omega(0) = \omega_0 & \quad ; \quad \omega(t) = 0 \end{aligned}$$

$$c \cdot \int_0^{\varphi_{\max}} \varphi_m \cdot d\varphi_m = J_m \cdot \int_{\omega_0}^0 \omega_m \cdot d\omega_m$$

$$\varphi_{\max} = \omega_0 \cdot \sqrt{\frac{J_m}{c}}$$

$$\frac{1}{2} \cdot c \cdot \varphi_m^2 \Big|_0^{\varphi_{\max}} = \frac{1}{2} \cdot J_m \cdot \omega_m^2 \Big|_{\omega_0}^0$$

$$c \cdot \varphi_{\max}^2 = -J_m \cdot \omega_0^2$$

Znak „-“ zbog usporenja (zanemaruje se)

## NAPOMENE:

$J_m$  - moment inercije svih translatornih i obrtnih delova motora redukovano na zamajac, uključujući i sam zamajac.

Najčešće se sa dovoljno tačnosti može uzeti da ovaj moment inercije iznosi:

$$J_m = 0.06 \div 1 \text{ kg/m}^2$$

ili da se odredi prema izrazu:

$$J_m = 1.02 \cdot m \cdot i^2 \text{ kg/m}^2$$

gde je:

m - masa zamajca

i - poluprečnik inercije

- **Više vrednosti inercionih momenata ostvaruju se u višim stepenima prenosa**, tj. kada je krutost transmisije veća, kao i pri većim ugaonim brzinama.
- Više vrednosti krutosti transmisije odgovaraju
  - **nižim vrednostima redukcije (viši stepeni prenos)** i
  - **manjim dužinama vratila**
- Limitirajući zaštitni uticaj **glavne frikcione spojnice**  
$$M_n = M_{e\max} \cdot \beta$$
- **Hidrodinamičke komponente** u transmisiji maksimalno relaksiraju inerciona opterećenja (**“klizanje” tečnosti – fluid je komponenta u rednoj vezi u toku snage**)

## PRIMER

**Vozilo:** FIAT 125     $P_{e\max} = 66.2 \text{ kW}$     pri     $\omega_e P_{e\max} = 524 \text{ s}^{-1}$

$$M_e = \frac{P_{e\max}}{\omega_e P_{e\max}} = \frac{66200}{524} = 126.3 \text{ Nm}$$

$$i_I = i_1 \cdot i_2 = 1.21 \cdot 2.83 = 3.422$$

$$c_I = 370 \text{ Nm/rad}$$

$$M_{jI} = 902 \text{ Nm}$$

$$i_{II} = i_1 \cdot i_2' = 1.21 \cdot 1.74 = 2.100$$

$$c_{II} = 668 \text{ Nm/rad}$$

$$M_{jII} = 1215 \text{ Nm}$$

$$i_{III} = i_1 \cdot i_2'' = 1.21 \cdot 1.12 = 1.361$$

$$c_{III} = 1927 \text{ Nm/rad}$$

$$M_{jIII} = 1850 \text{ Nm}$$

$$i_{IV} = 1.000$$

$$c_{IV} = 4016.2 \text{ Nm/rad}$$

$$M_{jIV} = 2970 \text{ Nm}$$

$\frac{M_{jI}}{M_e} = \frac{902}{126.3}$	$\frac{M_{jII}}{M_e} = \frac{1215}{126.3}$	$\frac{M_{jIII}}{M_e} = \frac{1850}{126.3}$	$\frac{M_{jIV}}{M_e} = \frac{2970}{126.3}$
$\frac{M_{jI}}{M_e} = 7.14$	$\frac{M_{jII}}{M_e} = 9.62$	$\frac{M_{jIII}}{M_e} = 14.65$	$\frac{M_{jIV}}{M_e} = 23.50$