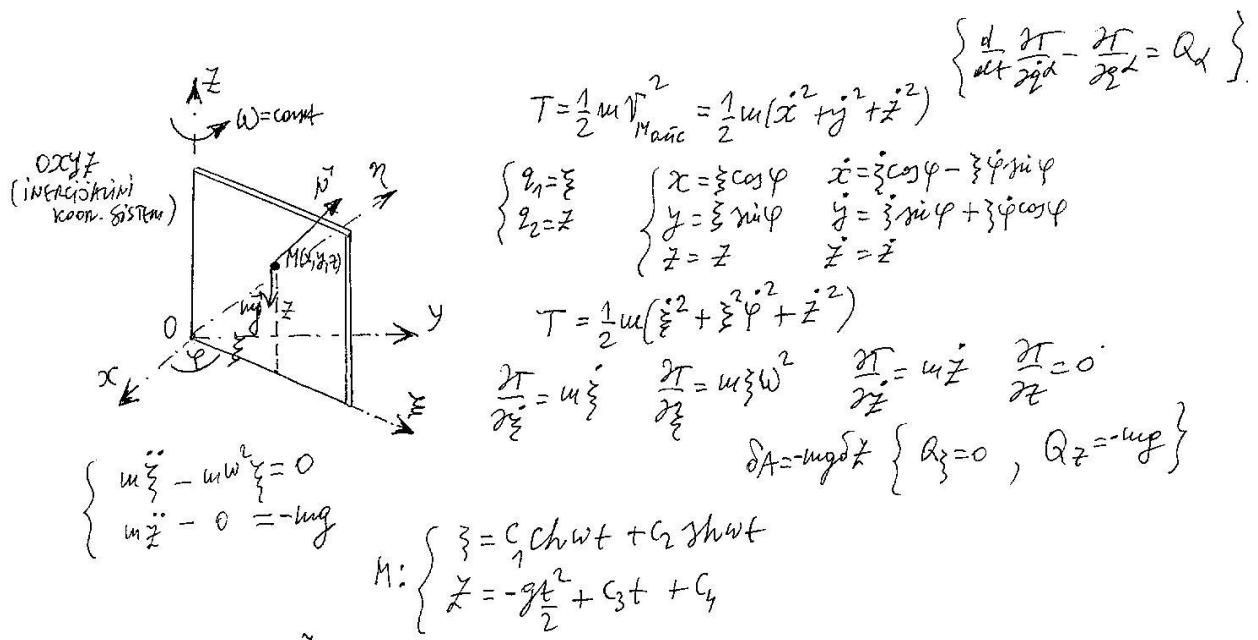


Primena Lagranževih jednačina druge vrste

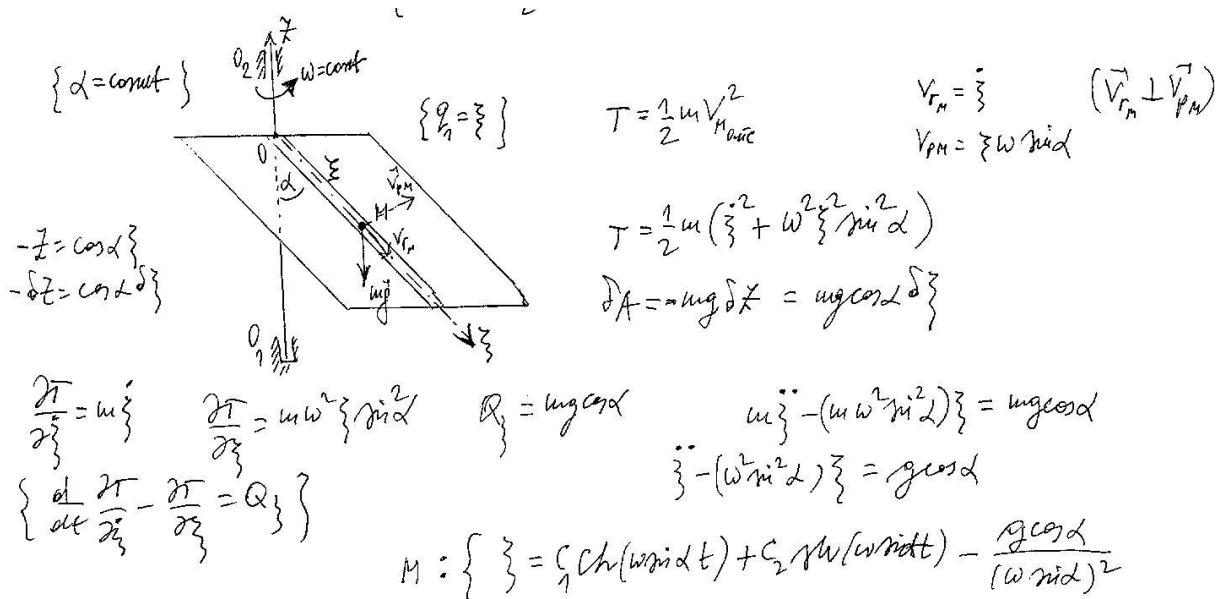
na

(1) relativno kretanje materijalne tačke:

1. Materijalna tačka M, mase m, može da se kreće bez trenja po ravni ξ_0z . Ravan ξ_0z obrće se oko nepomične ose oz konstantnom ugaonom brzinom ω . U početnom trenutku $t_0=0$, tačka M je bila u miru (u odnosu na ravan ξ_0z) u položaju $M_0(\xi_0=5, z_0=0)$. Odrediti opšte jednačine kretanja tačke M u odnosu na pokretni koordinatni sistem ξ_0z . Uzeti da su generalisane koordinate: ξ, z .



2. Ploča (u kojoj je izdubljen glatki kanal) zavarena je pod uglom α ($\alpha=\text{const}$) za vertikalnu osovinu i obrće se oko vertikalne nepomične ose 0_1O_2 konstantnom ugaonom brzinom ω . Kroz kanal može da se kreće tačka M mase m. Uzeti da je generalisna koordinata ξ (za kanal je vezana pokretna osa 0ξ), pa odrediti opštu jednačinu relativnog kretanja tačke M, tj. $\xi(t)=?$



3. Krivaja AD (AD=BE=R) obrće se konstantnom ugaonom brzinom ω i dovodi u kretanje cev DE (DE=2R, DO=OE=R). Veze u tačkama A, B, D i E su zglobne. Unutar glatke cevi DE može da se kreće tačka M mase m, vezana za dve opruge DM i ME, krutosti $c_1=2c$ $c_2=c$. U $t_0=0$ tačka M je u koordinatnom početku relativne koordinatne ose 0ξ s relativnom brzinom $R\omega_0$ (opruge su tada bile nenapregnute); odrediti opštu jednačinu relativnog kretanja tačke M, tj. $\xi(t)=?$ Uzeti da je generalisna koordinata ξ . Sistem je u vertikalnoj ravni, $\varphi(0)=0$.

Diagram illustrating the mechanical system. A horizontal beam AD is hinged at A and has a roller support at D. A vertical rod DE is hinged at D and has a roller support at E. A mass m is attached to the beam AD at point M and to the rod DE at point M. Two springs with stiffness c_1 and c_2 are attached to point M and to points D and E respectively. The beam rotates with angular velocity $\omega = \frac{c}{m}$. The rod DE rotates with angular velocity φ . The coordinate ξ is defined as the relative position of point M from the origin 0. The system is shown in a rotating coordinate system with axes x and y .

Initial conditions: $\varphi(0)=0$, $\dot{\varphi}(0)=0$, $\ddot{\varphi}(0)=0$, $\xi(0)=0$, $\dot{\xi}(0)=R\omega_0$.

Equations of motion:

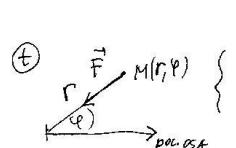
$$\begin{aligned} \ddot{\xi} + \frac{3c}{m}\xi &= R\varphi^2 \cos(\omega t) \quad (\text{Free vibration}) \\ \ddot{\xi} + \frac{3c}{m}\xi &= R\varphi^2 \cos(\omega t) \quad (\text{Damped vibration}) \end{aligned}$$

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na

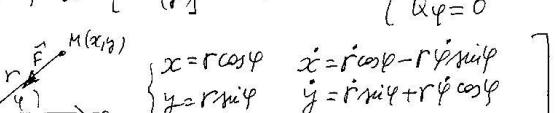
(2) kretanje materijalne tačke pod dejstvom centralne sile:

- Materijalna tačka M jedinične mase kreće se pod dejstvom centralne privlačne sile čiji je intenzitet $F = \frac{mk^2}{r^3}$. Odrediti za tačku M, u odnosu na date generalisane koordinate r, φ : 1) diferencijalne i konačne jednačine kretanja, 2) putanju. U početnom trenutku $t_0=0, r_0=2, \varphi_0=0, V_0=1/2$, a ugao između brzine i potega bio 45° . Neka je $k=1$; zadate veličine su date u osnovnim jedinicama SI sistema.

$\textcircled{1}$  $\left\{ \begin{array}{l} q_1 = r \\ q_2 = \varphi \end{array} \right.$

(1) $\frac{d}{dt} \frac{\partial T}{\partial \dot{q}_1} - \frac{\partial T}{\partial q_1} = Q_r \quad (2) \frac{d}{dt} \frac{\partial T}{\partial \dot{q}_2} - \frac{\partial T}{\partial q_2} = Q_\varphi \quad (u=1; k=1)$

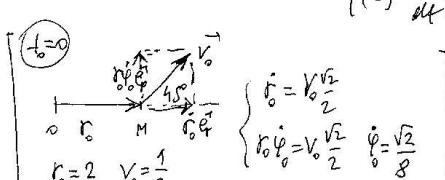
$dA = \vec{F} \cdot d\vec{r} = \left[\frac{f(r)}{F} \right] \cdot d\vec{r} = f(r) dr \quad \left\{ \begin{array}{l} Q_r = f(r) = -\frac{mk^2}{r^3} = -\frac{1}{r^3} \\ Q_\varphi = 0 \end{array} \right.$

$T = \frac{1}{2} u (\dot{r}^2 + r^2 \dot{\varphi}^2)$  $\left\{ \begin{array}{l} x = r \cos \varphi \\ y = r \sin \varphi \end{array} \right. \quad \left\{ \begin{array}{l} \dot{x} = \dot{r} \cos \varphi - r \dot{\varphi} \sin \varphi \\ \dot{y} = \dot{r} \sin \varphi + r \dot{\varphi} \cos \varphi \end{array} \right. \quad T = \frac{1}{2} u (\dot{x}^2 + \dot{y}^2) = \frac{1}{2} u (\dot{r}^2 + r^2 \dot{\varphi}^2)$

$\frac{\partial T}{\partial \dot{r}} = u \dot{r} \quad \frac{\partial T}{\partial r} = u r \dot{\varphi}^2 \quad \frac{\partial T}{\partial \dot{\varphi}} = u r^2 \dot{\varphi} \quad \frac{\partial T}{\partial \varphi} = 0$

(1) $u \ddot{r} - u r \dot{\varphi}^2 = -\frac{1}{r^3}$

(2) $\frac{d}{dt} (u r^2 \dot{\varphi}) - 0 = 0 \Rightarrow r^2 \ddot{\varphi} = r_o^2 \dot{\varphi}_o^2 = \frac{1}{2} \Rightarrow \dot{\varphi} = \frac{\sqrt{2}}{2r^2} \Rightarrow (1)$

$\textcircled{2}$  $\left\{ \begin{array}{l} \dot{r}_0 = \frac{V_0 \sqrt{2}}{2} \\ r_0 \dot{\varphi}_0 = V_0 \frac{\sqrt{2}}{2} \\ \dot{\varphi}_0 = \frac{\sqrt{2}}{8} \end{array} \right.$

$u=1 \quad \ddot{r} - r \left(\frac{\sqrt{2}}{2r^2} \right)^2 = -\frac{1}{r^3}$

$\ddot{r} = -\frac{1}{2r^3} \quad \ddot{r} = \frac{d}{dt} \frac{dr}{dt} \frac{dr}{dt} = \dot{r} \frac{d\dot{r}}{dr}$

$\int \dot{r} d\dot{r} = - \int \frac{dr}{2r^3} \quad \frac{\dot{r}^2}{2} = \frac{1}{4r^2} + C_1 \quad \frac{\dot{r}_0^2}{2} = \frac{1}{4r_0^2} + C_1 \Rightarrow C_1 = 0$

$\dot{r}^2 = \frac{1}{2r^2} \quad \dot{r} = \frac{1}{\sqrt{2r}} \quad \int r dr = \int \frac{dt}{\sqrt{2}} \quad \frac{\dot{r}^2}{2} = \frac{t}{\sqrt{2}} + C_2 \quad \frac{\dot{r}_0^2}{2} = 0 + C_2 \quad C_2 = 2$

$r^2 = \sqrt{2}t + 4 \quad \dot{\varphi} = \frac{\sqrt{2}}{2r^2} \quad \int d\varphi = \int \frac{r_0^2 dt}{2(\sqrt{2}t+4)} \quad \left\{ \begin{array}{l} \varphi = \frac{1}{2} \ln(2t + 4\sqrt{2}) \\ r = \sqrt{\sqrt{2}t + 4} \end{array} \right\} \Rightarrow \boxed{r^2 = e^{2\varphi}}$

2. Materijalna tačka M mase m kreće se pod dejstvom centralne privlačne sile čiji je intenzitet $F = \frac{mk^2}{r^7}$. Odrediti za tačku M, u odnosu na date generalisane koordinate r, φ : 1) diferencijalne i konačne jednačine kretanja, 2) putanju. U početnom trenutku $t_0=0, r_0=1, \varphi_0=0, V_0=1$, a ugao između brzine i potega bio 90° . Neka je $k^2=3$; zadate veličine su date u osnovnim jedinicama SI sistema.

$k=\sqrt{3}$

$$F = \frac{mk^2}{r^7}$$

$$(1) \frac{d}{dt} \frac{\partial T}{\partial \dot{r}} - \frac{\partial T}{\partial r} = Q_r \quad (2) \frac{d}{dt} \left(\frac{\partial T}{\partial \dot{\varphi}} \right) \left(\frac{\partial T}{\partial \varphi} \right)^0 = Q_\varphi \Rightarrow \frac{\partial T}{\partial \dot{\varphi}} = \text{const}$$

$$dA = f(r) dr \quad \begin{cases} Q_r = f(r) = -\frac{mk^2}{r^7} = -\frac{\sqrt{3}m}{r^7} \\ Q_\varphi = 0 \end{cases}$$

$$\begin{cases} V_0=1 & r_0=1 & \dot{\varphi}_0=0 \\ \dot{r}_0=0 & r_0\dot{\varphi}_0=V_0 \Rightarrow \dot{\varphi}_0=1 & \end{cases} \quad T = \frac{1}{2}m(\dot{r}^2 + r^2\dot{\varphi}^2) \quad \frac{\partial T}{\partial \dot{r}} = m\dot{r} \quad \frac{\partial T}{\partial r} = mr\dot{\varphi}^2 \quad \frac{\partial T}{\partial \dot{\varphi}} = m\dot{\varphi}r^2 \quad \frac{\partial T}{\partial \varphi} = 0$$

$$\begin{cases} (1) m\ddot{r} - mr\dot{\varphi}^2 = -\frac{m(\sqrt{3})^2}{r^7} & \ddot{r} - r\left(\frac{1}{r^2}\right)^2 = -\frac{(\sqrt{3})^2}{r^7} \\ (2) r^2\ddot{\varphi} = r_0^2\dot{\varphi}_0 = 1 & \dot{\varphi} = \frac{1}{r^2} \end{cases} \Rightarrow \int \dot{r} dr = \int \left(\frac{1}{r^3} - \frac{3}{r^7} \right) dr$$

$$\frac{\dot{r}^2}{2} = -\frac{1}{2r^2} + \frac{1}{2r^6} + c_1 \quad (\dot{r}_0=0) \quad \dot{r}^2 = \frac{(1-r^4)}{r^6} \Rightarrow \int \frac{r^3}{\sqrt{1-r^4}} dr = \int dt$$

$$r^4 = 1 - 4t^2 + c_2 \quad (r_0=1-0+c_2 \Rightarrow c_2=0)$$

$$(2) \Rightarrow \int \omega \varphi = \int \frac{dt}{\sqrt{1-4t^2}}$$

$$\begin{cases} r^4 = 1 - 4t^2 \\ \varphi = \frac{1}{2} \arcsin(4t) + c_3 \rightarrow 0 \\ m(2\varphi) = 2t \quad r^4 = 1 - (2m\varphi)^2 \Rightarrow \end{cases} \Rightarrow \boxed{r^2 = \cos 2\varphi}$$